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Fatigue Properties of Nonferrous Alloys for Heat Exchangers, Pumps, and Piping

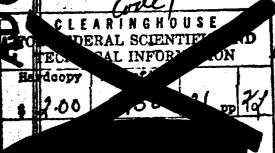
> Assignment 86 108 MEL R&D Report 232/66 May 1966

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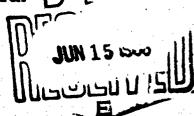
M. R. Gross and R. C. Schwab

U. S. NAVY

# MARINE ENGINEERING LABORATORY



Annapolis, Md/DD



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Approved by

Naval Alloys

#### **ABSTRACT**

The fatigue behavior of 13 nonferrous alloys used for corrosion-resistant heat exchangers, pumps, and piping systems was investigated over a broad life spectrum of 100 to 100-million cycles. Both cast and wrought copper-base and nickel-base alloys were studied. It is concluded that wrought Monel\* and forged Ni-Al bronze have the highest fatigue strengths, whereas gun metal and valve bronze have the lowest. The effect of salt water on fatigue performance was not found to be highly significant. The use of Langer's equation to predict stress-cycle relationships gave satisfactory results for wrought alloys but appeared to be overly conservative for cast alloys.

<sup>\*</sup>Registered trade name of the International Nickel Company, Incorporated.

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	ckel 707 (Wrought)	
	eve, Flexural Fatigue Curves, Monel	
	(Cast)	
	ve, Flexural Fatigue Curves, Monel	
·	ought)	

#### Introduction

Many copper-base and nickel-base alloys are used in the construction of heat exchangers, pumps, and piping systems designed to handle fresh or saline water. In the selection of materials for such applications, consideration is given primarily to corrosion resistance, erosion resistance, and heat transfer characteristics. In most applications the applied stress levels are low. Accordingly, the structural strength properties of the materials are relatively unimportant.

In recent years, more and more attention has been given to the structural properties of these alloys because of (1) cost and weight reduction programs, (2) conservation of strategic materials, (3) development of new high-strength alloys, and (4) new applications which impose high stress levels. Typical of the latter are sea-connected cooling systems for hydrospace vehicles.

One of the most likely modes of mechanical failure in systems undergoing cyclic pressurization or thermal shock loading is metal fatigue. The frequency of stress cycling in such systems may vary from that of an occasional start-up and shutdown to vibrational forces developed by the movement of the heat-exchanger fluids. Little or no published information on the alloys used in this type of service was found in reviewing the literature several years ago. Accordingly, tests were conducted at the U. S. Navy Marine Engineering Laboratory to establish the fatigue behavior of

ariety of corrosion-resistant nonferrous alloys. The results these tests are presented in this paper.

### .erials Investigated

The 13 alloys investigated are listed in Table 1, together their chemical compositions and tensile properties. !luded are the strength coefficient, K, and strain-hardening soment, n, contained in the true-stress/true-strain relation-up:

$$\sigma = \kappa \epsilon^n$$

ere are some deficiencies in the tensile properties of the cast terials with respect to the governing specifications. This is be expected inasmuch as the specification requirements are tally based on separately cast test coupons, whereas the values wen in Table 1 were obtained on specimens removed from cast ates.

#### thod of Test

Two types of flexural fatigue specimens were used in the vestigation. The high-cycle fatigue tests were performed with tating cantilever-beam specimens having the dimensions shown in gure 1. These were constant deadweight load tests with a cycle equency of 1450 cpm. The smooth test lengths were circumrentially and longitudinally polished to a metallographic nish.

Chemical Composition and Mechanical Properties of Alloys Table 1

	L													X.	Mechanical		Prope	Properties		
		-											SX.				-	-	-	
			Spacifi			ŧ	, C. M.		4				off-				,	Hè	1	EI V
Alloy	Type	Type Condition		20	Ni	Fe	An		No.	ع ا	. 5	Ot here	set)	2 3	# .E	ڊ لا ≩	× '		×	100
Gun Metal (Comp G)		As-cast	Ξ	37.2	2.0	0.02	,	3.5		<b>9.4</b>	_	1	15.6	_	_	4	-	0.45	9 <u>9</u>	Ps.1
Valve	U	As-cast	MIL-B-16541 88.	88.5	0.5	0.01	1.	3.8		5.6	1.	Pb-1.6	15.8	28.0	13/	20	580.	0.34	39	17
Gronze (Comp M)												P-0.01								1
Ni-Al Bronze	ပ	As-cast	MIL-B-21230 30.1	30.1	5.2 5.7	3.7	0.7	,	10.3		<u> </u>		42.1	97.3	1,4	161	181 0.	0.32	8	13
Ni-Al Bronze	Çi.	Annealed	OC-B-679 Comp 2	81.2	4.5	2.8	6.0	·	10.6	Nil		1.	51.8	103.3	16	15 2	206 0.28		66	17
Superston 40	ပ	As-cast	21250 2	9.4%	2.2	3.3	12.5	,	7.4			-	43.4	85.4	50	24 1	154 0.	0.26	82	118
70-30 Cupro- nickel	ပ	As-cast	MIL-C-20159		30.4	5.0	1.25	,	1	,	0.5	5°0 q2	48.1	79.2	23	39 1	151 6.	0.27	22	13
70-50 Cupro- nickel	3	Annealed	MIL-C-15726 63.6		29.6 0.6	0.6	6.0	0.2	•	'	ı	,	20.2	58.5	642	701	116 0.21		28	22
90-10 Cupro- nickel	3	Hard	MIL-C-15726 88.0	<u> </u>	10.0	1.3	0.0	0.04	ı	,		40.02	49.6	53.8	ñ	1	73 0.09		69	50
Cufenloy- 40	3	Annealed			Rem	2.2	1.4	90.0	0.01	0.01		Pb-0.01	22.4	70.3	20	78 1	174 0.	0.48	59	54
Cufenloy-	3	DSR*	•			2.2	1.4	90.0	0.01	0.01		+	68.4	79.0	26	72 1	110 0.10	<u>ان</u> ــــــــــــــــــــــــــــــــــــ	8	20
, – ,	3	Annealed		64.1	29.6	7.5	0.8	0.11	1	<0.05 0.02	<del></del>	+	51.0	81.6	8	48	148 0.25	<del> </del>	83	22
Monel "E"	υ	' 1	00-N-288 Class "E"	-	63.1	2.2	0.9	,	ı	,	9.7	c-0.1	27.0	59.5	23	24	24 128 0.36		102	23
Tauou		Annealed	IL-N-894 lass A	25.7	0.50	6.0	1.0	-	0.02			1	33.1	83.6	9#	<u>-</u>	70 167 0.35		36	36
*Draw word		Stress relieved	Ţ											1		1	1	_	-	

\*Drawn and Stress relieved.
C - Cast; F - Forged; W - Wrought
YS - Yield Strength
TS - Tensile Strength
Elong - Elongation
RA - Reduction of Area

The low-cycle fatigue tests were performed with equipment ibed previously. If lat flexure-type specimens having the sions shown in Figure 2 were used. The short end of the men was held stationary, while the long end was flexed en mechanical stops by a hydraulic piston. One or more n gages (0.25-inch gage length) were attached to the miniest section to record the longitudinal strain. The applied ng force was measured with a load cell. The total strain ,  $\Delta \varepsilon_{\rm T}$ , was obtained from strain gage readings, and the al bending-stress range was calculated from elastic stress las using the measured load range. Specimens were cycled cpm.

All of the fatigue tests were of the completely reversed (Fatigue ratio = -1). Whereas most of the specimens were d in air, a few were tested with Severn River water conusly wetting the test surface. Severn River water is a ish estuary water containing 1/6 to 1/3 the salt content tural seawater, depending on the season and the tide. ous fatigue tests in both Severn River water and natural ter have shown no significant differences in the effects e two media.

#### re Criteria

Failure in the high-cycle, rotating cantilever-beam tests sted of complete fracture. Failure in the low-cycle fatigue

tests was defined as one or more surface cracks 3/16 to 1/4 inch in length.

#### Results of Tests

The results of the tests are plotted in log-log form in Figures 3 through 15. Two methods have been used in analyzing the data. The top graph in each figure is the  $S_R$  vs N relationship for the data, where  $S_R$  is the nominal reversed bending stress and N is the number of cycles to failure.  $S_R$  was calculated from the elastic stress formula

$$S_{R} = \frac{\Delta Mc}{2I} \qquad \dots (1)$$

where  $\Delta M$  = bending moment range, in-lb.

c = distance from neutral axis to outermost fiber at minimum cross section, in.

I = moment of inertia of minimum cross section, in.4

The bottom graph in each figure is the  $S_{\text{PE}}$  vs N relationship for the data, where  $S_{\text{PE}}$  is the reversed pseudoelastic or apparent elastic stress calculated as follows:

$$S_{PE} = \frac{\Delta \varepsilon_{T} \cdot E}{2} \qquad .... (2)$$

where  $\Delta \varepsilon_{T}^{}$  = total strain range as determined from strain gages on the test section, in/in.

E = modulus of elasticity (Table 1), psi.

urvilinear relationship between  $S_{pr}$  and N was obtained by fitting the following relationship to the data.

$$S_{PE} = \frac{C}{N^{m}} + S_{E} \qquad .... (3)$$

C and m = best-fit constants

 $S_E$  = endurance limit or fatigue strength at  $10^8$  cycles, psi.

est-fit equation for the  $S_{\text{PE}}$  vs N data is given in each  $\bullet$ .

Equation (3) is a generalization of the following equation sed by Langer<sup>2</sup> for predicting the  $S_{\text{PE}}$  vs N fatigue curve tensile test data.

$$S_{PE} = \frac{E}{4N^{0.5}} \ln \left( \frac{100}{100-RA} \right) + S_{F}$$
 .... (4)

RA = reduction of area, percent.

The dashed line in each figure is Langer's predicted curve on Equation (4) and the tensile data presented in Table 1.

The triangle symbols in graphs represent specimens which seen continuously exposed to salt water during the fatigue

### cison of Fatigue Strengths

Table 2 represents an attempt to rationalize the fatigue for the 13 alloys investigated. The  $S_R$  and  $S_{PE}$  values were from the curves in Figures 3 through 15. The values in solumn were then ranked in order of decreasing fatigue

strength. An average rank for each material is shown in the right hand column.

Table 2

Comparison of Fatigue Strengths of Alloys Investigated

<u> </u>		<u> </u>		F	atigue	Strengt			
				Cycles		Cycles	10 <sup>8</sup> C		
Alloy	Type	Condition	S <sub>R</sub> ksi	Sp <u>e</u> ksi	S <sub>R</sub> ksi	SpE ksi	S <sub>R</sub> ksi	S <sub>PE</sub> ksi	Average Rank
Gun Metal	Cast	As-cast	35 (12)	96(12)	17(12)	25(12)	6(12.5)	8(12.5)	(12.2)
Valve Bronze	Cast	As-cast	32(13)	52(13)	16(13)	17(13)	6(12.5)	8(12.5)	(12.8)
Ni-Al Bronze	Cast	As-cast	78(6)	210(5)	46(4)	48(7)	29(3)	30(3)	(4.7)
Ni-Al Bronze	Forged	Annealed	110(1)	230(3)	65(1)	64(1)	35(1.5)	37(1)	(1.4)
Superston 40	Cast	As-cast	83(4)	230(3)	44(5)	50(5)	25(5.5)	25(5.5)	(4.7)
70-30 Cupronickel	Cast	As-cast	70(8)	130(11)	32(10)	27(11)	13(11)	14(11)	(10.3)
70-30 Cupronickel	Wrought	Annealed	57(11)	180(7)	29(11)	54(2)	25(5.5)	25(5.5)	(7.0)
90-10 Cupronickel	Wrought	Hard	74(7)	160(8.5)	38(7)	40(8)	21(8)	21(9)	(7.9)
Cufenloy 40	Wrought	Annealed	58(10)	230(3)	35(8)	38(9)	26(4)	26(4)	(6.3)
Cufenloy 40	Wrought	DSR	100(2)	160(8.5)	48(3)	50(5)	20(9)	23(7.5)	(5.8)
Cupronickel-707	Wrought	Annealed	90(3)	190(6)	50(2)	50(5)	22(7)	23(7.5)	(5.1)
Monel "E"	Cast	As-cast	62(9)	155(10)	34(9)	36(10)	16(10)	16(10)	(9.7)
Mone1	Wrought	Annealed	80(5)	350(1)	40(6)	52(3)	35(1.5)	33(2)	(3.1)

Note: Numeral in ( ) is rank of value.

### Conclusions

From the data and curves presented in Figures 3 through 15, the following conclusions have been reached relative to the unnotched fatigue behavior of the materials investigated.

 Variations in fatigue strength or life are greater for cast alloys than for wrought alloys.

- The fatigue strength of a wrought alloy is superior to t of a cast alloy of comparable chemical composition.
- Wrought Monel and forged Ni-Al bronze have the highest igue strengths, whereas gun metal and valve bronze have the est.
- Stress-cycle relationships predicted by Langer's equation generally satisfactory for wrought alloys but appear to be rly conservative for most cast alloys.
- Salt water does not have a highly significant effect on fatigue behavior of the alloys investigated.

#### erences

Gross, M. R., "Low-Cycle Fatigue of Materials for Submarine Construction," <u>Naval Engineers Jour</u>, Vol. 75, No. 5, Oct 1963, pp. 783-797

Langer, B. F., "Design of Pressure Vessels for Low-Cycle Fatigue," Jour of Basic Engineering, ASME Trans. Ser. D, Vol. 84, Series D, No. 3, Sep 1962, pp. 389-402

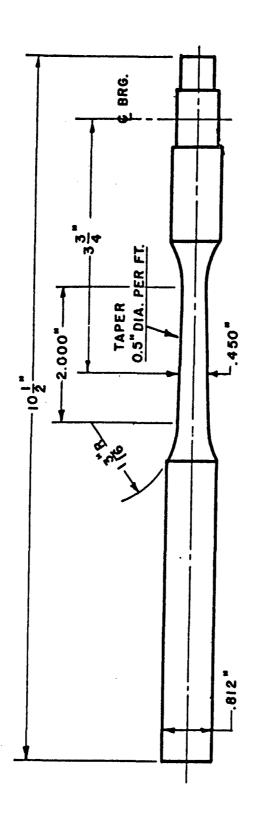


Figure 1

Rotating Cantilever Beam Fatigue Specimen

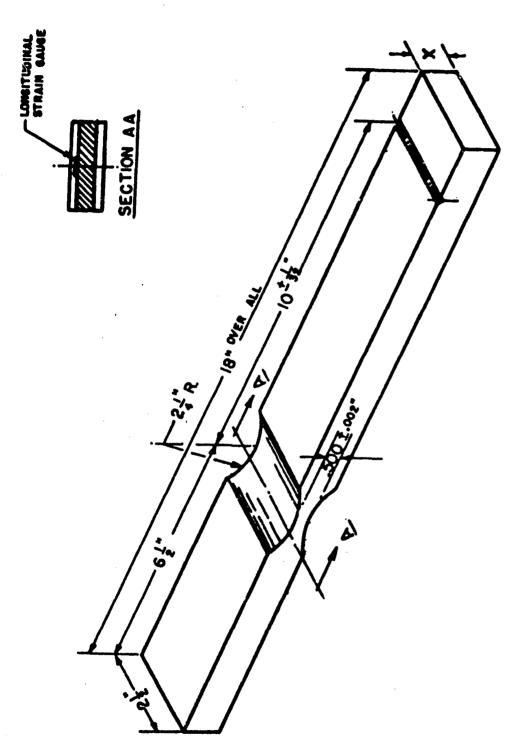


FIGURE 2 - LOW-CYCLE FATIGUE SPECIMEN

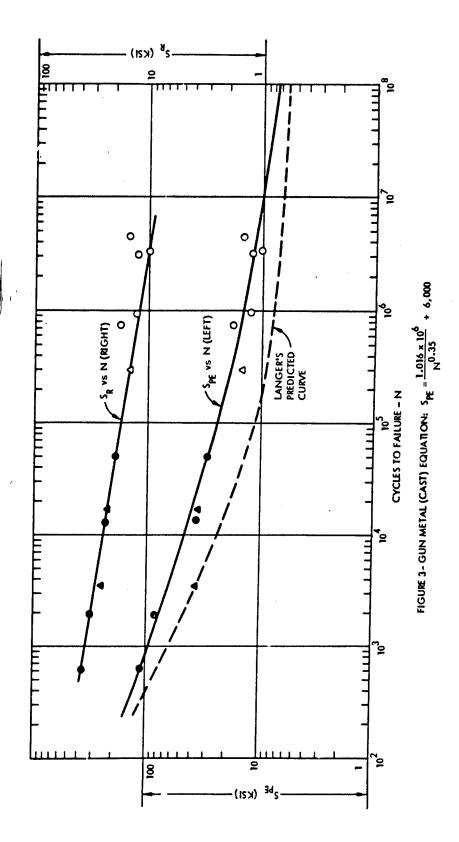
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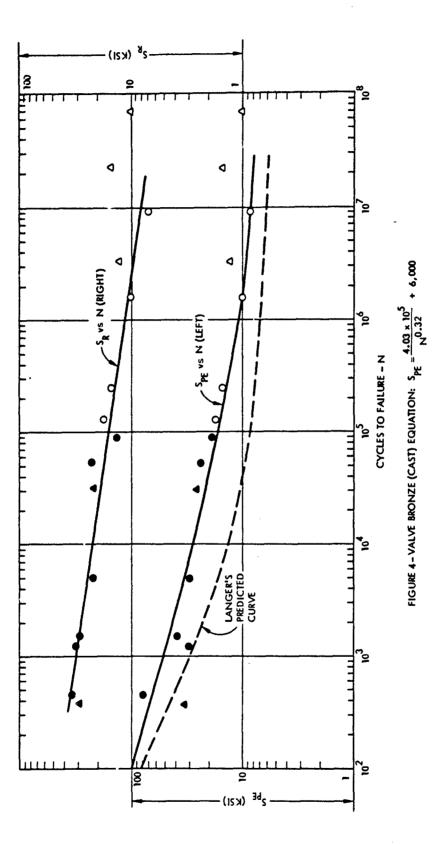
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### Flexural Fatigue Curves

### Legend

- O Rotating Cantilever Fatigue Tests, Air
- $\Delta$  Rotating Cantilever Fatigue Tests, Salt Water
- - Low-Cycle Fatigue Tests, Air
- ▲ Low-Cycle Fatigue Tests, Salt Water





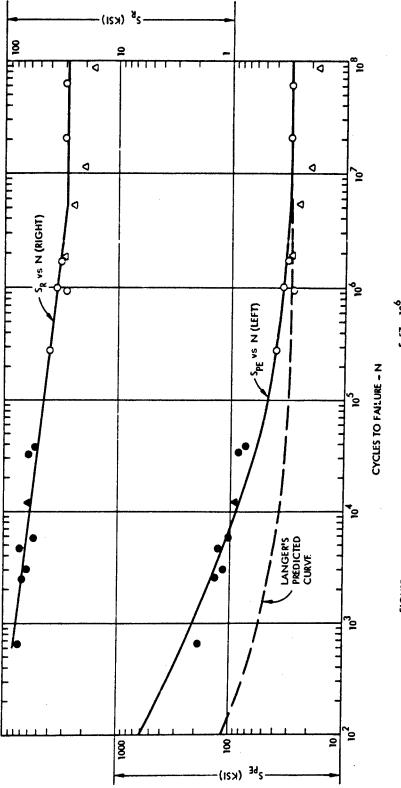


FIGURE 5 - Ni-A1 BRONZE (CAST) EQUATION:  $S_{PE} = \frac{5.57 \times 10^6}{N^{0.49}} + 29,000$ 

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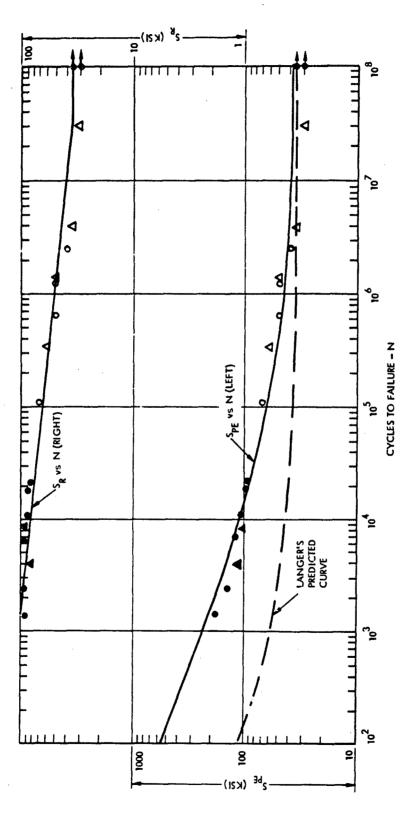
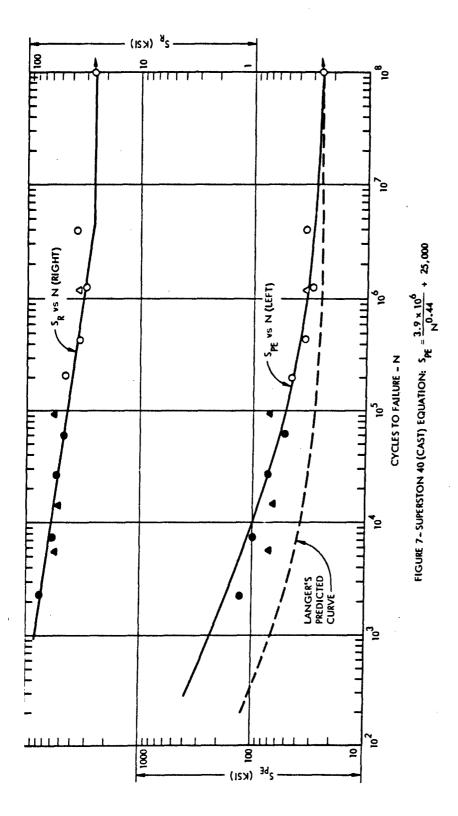
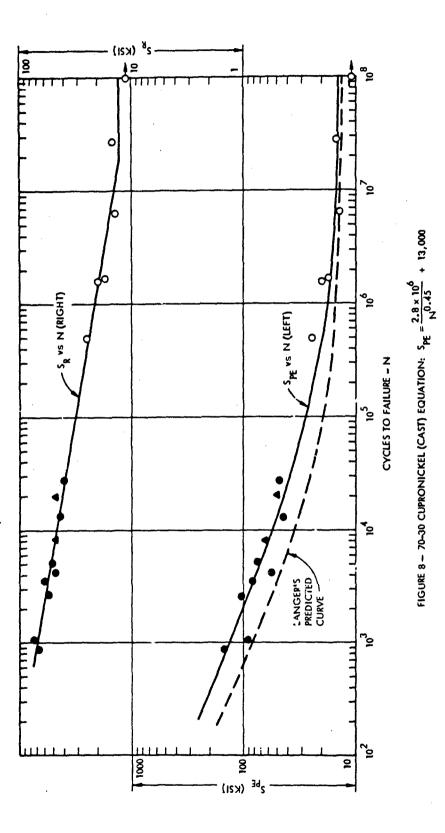


FIGURE 6 - Ni-A1 BRONZE (FORGED) EQUATION:  $s_{PE} = \frac{3.53 \times 10^6}{N^{0.42}} + 35,000$ 





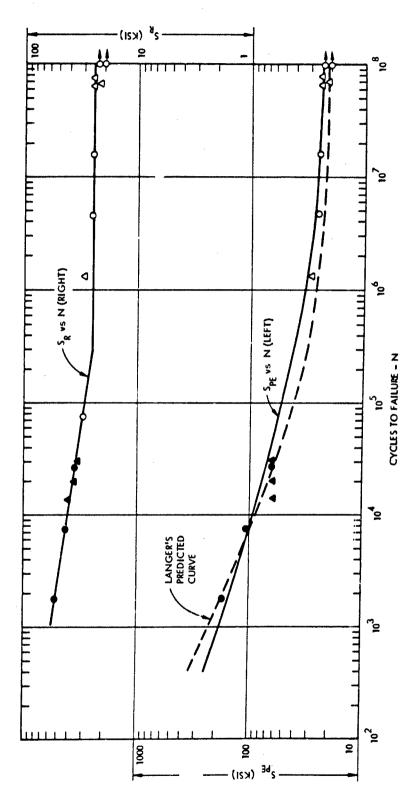
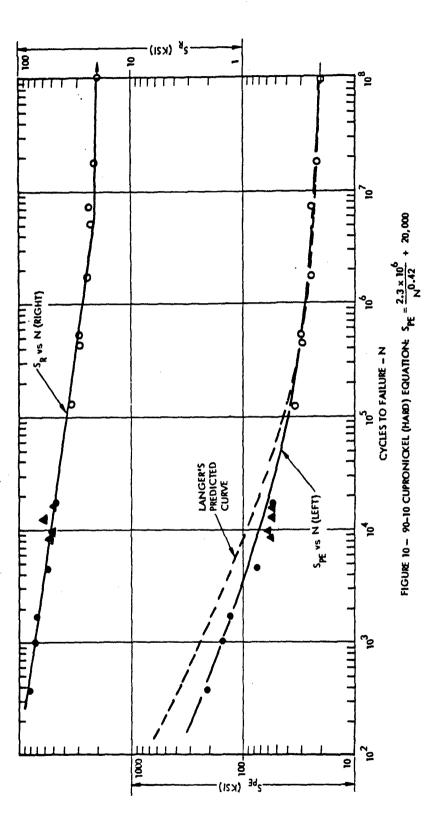
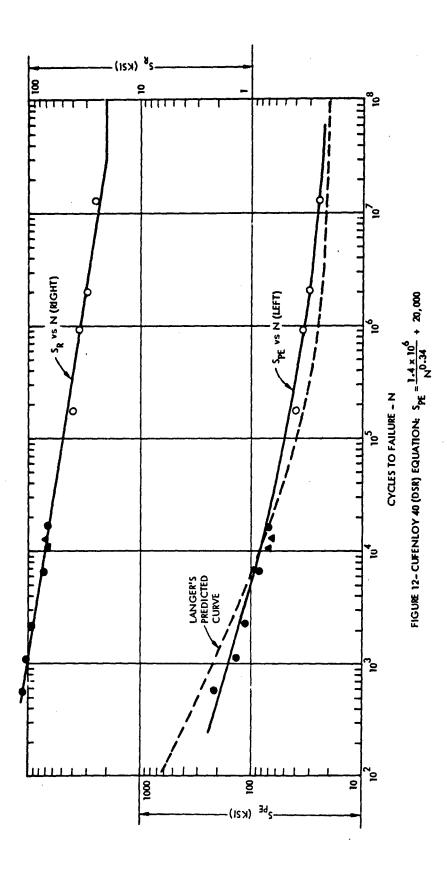


FIGURE 9 – 70-30 CUPRONICKEL (WROUGHT) EQUATION:  $S_{PE} = \frac{1.90 \times 10^6}{N^{0.35}} + 20,000$ 



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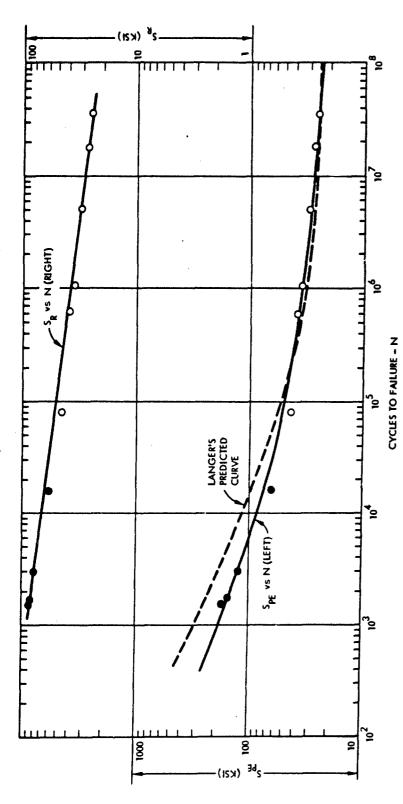
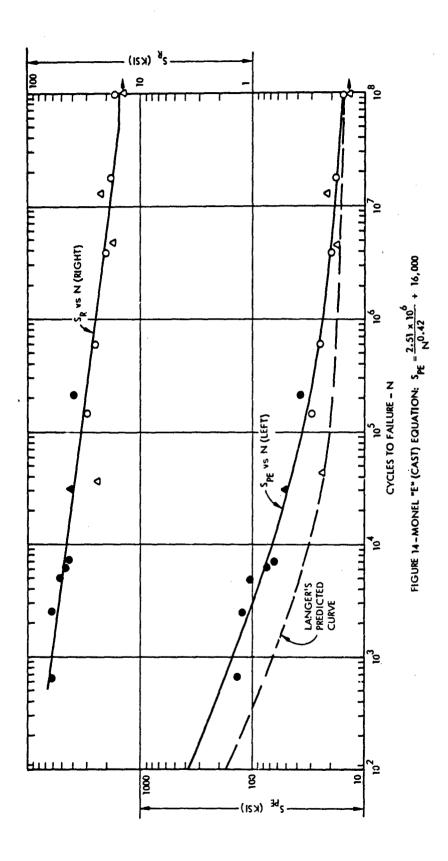


FIGURE 13.- CUPRONICKEL 707 (WROUGHT) EQUATION:  $s_{\text{PE}} = \frac{2.4 \times 10^6}{\text{N}^{0.39}} + 25,000$ 

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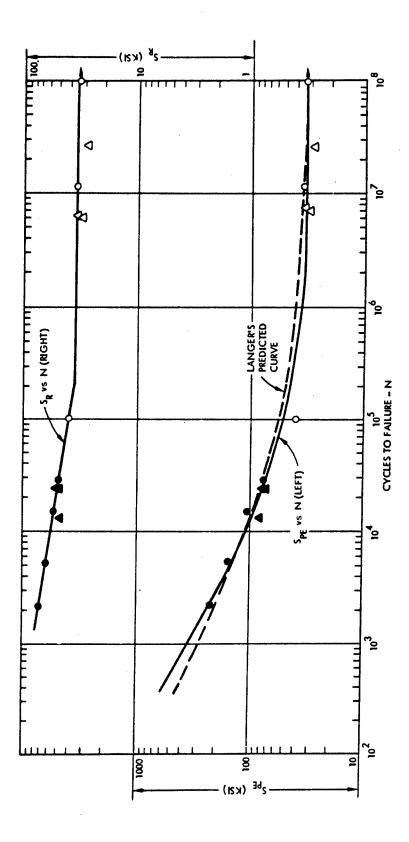


FIGURE 15-MONEL (WROUGHT) EQUATION:  $S_{PE} = \frac{2.1 \times 10^7}{N^0.61}$ 

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Piping systems								
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There is no limitation on the length of the abstract. However, the suggested length is from 150 to 225 words.

14. KEY WORDS: Key words are technically meaningful terms or short phrases that characterize a report and may be used as index entries for cataloging the report. Key words must be selected so that no security classification is required. Identifiers, such as equipment model designation, trade name, military project code name, geographic location, may be used as key words but will be followed by an indication of technical context. The assignment of links, rales, and weights is optional.